# Gestion de l'énergie des véhicules hybrides avec prise en compte des dynamiques thermiques

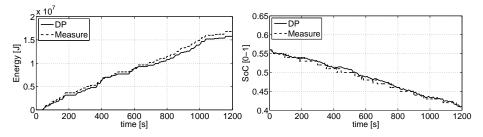
A. Sciarretta, D. Maamria IFP Energies nouvelles

### Journées du Groupe de Travail Automatique et Automobile

06/11/2014



- The calculation of an appropriate energy management system (EMS) for HEV can be formulated as an optimal control problem (OCP)
- Optimal online energy management has been proven for HEVs and PHEVs
- Example: IFPEN democar, ECMS implemented
- Experimental traces of fuel consumption and SOC can be very close to optimal
- In standard optimal EMS, only the SOC is considered as a dynamic variable in the OCP





- However, thermal dynamics (of the power-train components, thermal accessories, heat recuperation, etc.) are comparably slow as that of SOC
- and they can be important to take into account because they affect fuel consumption
- but also other criteria (aging, pollutant emissions)
- Extended optimal EMS can be solved off line using either DP or PMP
- Include such dynamics in online optimization as it has been done in ECMS for the SOC costate is a new challenge
- $\Rightarrow$  general problem of online optimal control of multi-state systems (with unknown perturbations)

## Engine temperature: Motivation

- Most EMS designs described in the literature assume that the engine is under thermal equilibrium (engine temperature is around 80°C)
- In some situations, thermal transients are not negligible:
  - the engine is subject to stop-start phases
  - engine temperature impacts emission and fuel consumption rates
  - the efficiency of after-treatments systems is relatively poor at low temperatures

### Objectives

- Quantify the benefit of including engine temperature in the EMS minimizing fuel consumption
- Find a sub-optimal, real-time capable approach

# Engine temperature: OCP Formulation

• The cost function to be minimized is

$$J(u) = \int_0^T c(u, w) e(\theta_e) dt$$

- c(.) is the fuel consumption rate when engine is warm
- e(.) is the correction factor of fuel consumption
- *u* is the engine torque
- w is the uncontrolled disturbance (drive cycle, etc.)
- Two state variables, SOC ( $\xi$ ) and a lumped engine temperature  $heta_e$

$$\dot{\xi} = f(u, w), \quad \dot{\theta_e} = g(u, w, \theta_e)$$

• Global constraints:  $\xi(0) = \xi_0$ ,  $\xi(T) = \xi_f$ ,  $\theta_e(0) = \theta_0$ ,  $\theta_e(T) = \text{free}$ 

## Engine temperature: OCP Solution using PMP

• Hamiltonian definition

$$H_1(u, w, \theta_e, \lambda, \mu) = c(u, w)e(\theta_e) + \lambda \frac{d\xi}{dt} + \mu \frac{d\theta_e}{dt}$$

• Euler–Lagrange equations

$$\begin{split} \dot{\lambda}(t) &= -\frac{\partial H_1}{\partial \xi}(u^*(t), w, \theta_e, \lambda, \mu) = 0\\ \dot{\mu}(t) &= -\frac{\partial H_1}{\partial \theta_e}(u^*(t), w, \theta_e, \lambda, \mu), \quad \mu(T) = 0 \end{split}$$

• Optimal control  $u^*$ 

$$u^*(t) = \arg\min_{u \in U^{ad}} H_1(u, w, \theta_e, \lambda, \mu)$$

 $\bullet$  Initial values  $\lambda(0),\,\mu(0)$  found by shooting methods

# Engine temperature: OCP Results and Comparison

- Looking for a methodology to include engine temperature in real-time EMS
- Proposed solution: Control model (Hamiltonian) simplification
- Calculate the optimal controls solutions of

$$S : u^{*}(t) = \arg \min_{u \in U^{ad}} \underbrace{ \begin{bmatrix} c(u, w)e(\theta_{e}) + \lambda \frac{d\xi}{dt} + \mu \frac{d\theta_{e}}{dt} \end{bmatrix}}_{H_{1}(u, w, \theta_{e}, \lambda, \mu(t))}$$

$$S_{1} : u^{*}_{1}(t) = \arg \min_{u \in U^{ad}} \underbrace{ \begin{bmatrix} c(u, w) + \lambda \frac{d\xi}{dt} \end{bmatrix}}_{H_{1}(u, w, \theta_{e} = \theta_{w}, \lambda, \mu(t) \equiv 0)}$$

$$S_{2} : u^{*}_{2}(t) = \arg \min_{u \in U^{ad}} \underbrace{ \begin{bmatrix} c(u, w)e(\theta_{e}) + \lambda \frac{d\xi}{dt} \end{bmatrix}}_{H_{1}(u, w, \theta_{e}, \lambda, \mu(t) \equiv 0)}$$

• Evaluate the real fuel consumption J(.) and calculate the state trajectories for each control:  $u^{\ast}$  and  $u^{\ast}_i,\ i=1,2$ 

## Engine temperature: OCP Results and Comparison

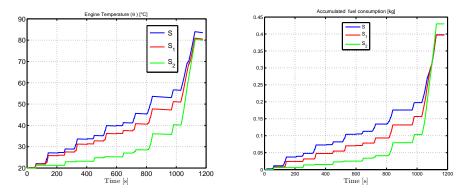


Figure: Optimal engine temperature trajectories for NEDC.

Figure: Accumulated fuel consumption trajectories for NEDC.

# Engine temperature: OCP Results and Comparison

Cycle	$S_1$	$S_2$	S
NEDC	4.80	5.20	4.79
Cycle 1	4.49	5.4	4.46
Cycle 2	5.34	5.71	5.32

### Table: Fuel consumption in [L/100km].

- $\bullet\,$  Error in fuel consumption between S and  $S_1$  less than 0.5%
- Error in fuel consumption between S and  $S_2$  higher  $\Rightarrow$  importance of a correct simplification of the control model

### Conclusion

Neglecting engine temperature changes in the EMS design leads to an acceptable sub-optimal solution (less than 0.5 %)  $\Rightarrow$  Simplify the numerical method used to solve the OCP.

# Engine temperature: Toy Problem

- Motivation: Justify and understand the previous result of model simplification.
- Toy problem considered
  - Cost function:

$$J(u) = \int_0^T (\frac{1}{2}au^2 - b\theta u)dt$$

• Dynamics:

$$\frac{d\xi}{dt} = r_0(D-u)$$

$$\frac{d\theta}{dt} = cu$$

Boundary conditions:

$$\xi(0) = \xi_0 \quad \xi(T) = \xi_0$$
$$\theta(0) = \theta_0, \theta(T) = \text{free}$$

Engine temperature

Catalyst Temperature

Theoretical Justification

## Engine temperature: Toy Problem

$$(S_{1}) \qquad \begin{cases} H_{1}(u,\lambda) = \frac{1}{2}au^{2} - b\theta_{\infty}u + \lambda\frac{d\xi}{dt} \\ \lambda = \frac{a}{T}E - b\theta_{\infty} \\ u_{0}^{*} = \frac{E}{T} \\ J(u_{0}^{*}) = \frac{a}{2T}E^{2} - b(\theta_{0} + \frac{c}{2}E)E \\ \end{cases}$$

$$(S) \qquad \begin{cases} H(u,\lambda,\theta,\mu) = \frac{1}{2}au^{2} - b\theta u + \lambda\frac{d\xi}{dt} + \mu\frac{d\theta}{dt} \\ \lambda = \frac{a}{T}E - b(\theta_{0} + cE), \quad \mu(t) = -bE + \frac{bE}{T}t \\ u^{*} = \frac{E}{T} \\ J(u^{*}) = \frac{a}{2T}E^{2} - b(\theta_{0} + \frac{c}{2}E)E \end{cases}$$

• Same controls, same state trajectories and same cost with different adjoint states.

## Battery temperature: Motivation

- Standard formulation of ECMS, etc. minimizes only fuel consumption
- Battery temperature is a key factor (alongside to current) for battery aging:
  - capacity loss  $\Rightarrow$  replacement costs, operational costs
  - resistance increase  $\Rightarrow$  operational costs
- Multi-objective optimal control formulation with minimization of a mixed fuel-aging cost
- Thermal dynamics and a model of aging mechanisms to be considered in the optimal control

### Objectives

- Quantify the benefit of including battery temperature in the EMS minimizing a trade off between fuel consumption and battery aging
- Find a sub-optimal, real-time capable approach

## Battery temperature: OCP Formulation

• The combined cost function to be minimized is

$$J = (1 - \alpha) \int_0^T c(u, w) \cdot LHV \, dt + \alpha \int_0^T \beta \cdot \dot{Y}(I(u, w), \xi, \theta_{bat}) \, dt$$

- Y is aging rate
- $\alpha$  is a weighting factor to adjust the two criteria
- $\beta$  is an arbitrary transformation coefficient
- $\bullet\,$  Two state variables, SOC  $\xi$  and a lumped battery temperature  $\theta_{bat}$

$$\dot{\xi} = f(I(u, w)), \quad \dot{\theta}_{bat} = f_{\theta}(I(u, w), \theta_{bat})$$

• Global constraints:  $\xi(0) = \xi_0$ ,  $\xi(T) = \xi_f$ ,  $\theta_{bat}(0) = \theta_0$ ,  $\theta_{bat}(T) = \text{free}$ 

# Battery temperature: OCP Solution using PMP

• Hamiltonian definition

$$\begin{split} H(u, w, \xi, \theta, \lambda_{\xi}, \lambda_{\theta}) &= (1 - \alpha) \cdot c(u, w) \cdot LHV + \alpha \cdot \beta \cdot \dot{Y}(u, w, \xi, \theta) + \\ &+ \lambda_{\xi} P_{ech}(u, w, \xi, \theta) + \lambda_{\theta} P_{th}(u, w, \xi, \theta) \end{split}$$

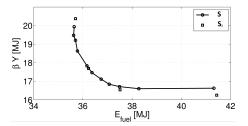
• Euler–Lagrange equations

$$\begin{aligned} C_{nom} U_0 \dot{\lambda}_{\xi} &= \frac{\partial H}{\partial \xi} = \alpha \cdot \beta \cdot \frac{\partial \dot{Y}}{\partial \xi} + \lambda_{\xi} \frac{\partial P_{ech}}{\partial \xi} + \lambda_{\theta} \frac{\partial P_{th}}{\partial \xi} \\ MC \dot{\lambda}_{\theta} &= \frac{\partial H}{\partial \theta} = \alpha \cdot \beta \cdot \frac{\partial \dot{Y}}{\partial \theta} + \lambda_{\xi} \frac{\partial P_{ech}}{\partial \theta} + \lambda_{\theta} \frac{\partial P_{th}}{\partial \theta} \end{aligned}$$

- Initial values  $\lambda_{\xi}(0)$ ,  $\lambda_{\theta}(0)$  found with shooting algorithms
- Control model simplification: compare
  - Two-states strategy (S)
  - SOC-state-only strategy (S<sub>1</sub>), with  $\lambda_{\theta} \equiv 0$  ( $\theta_{bat}$  considered as a constant).

# Battery temperature: OCP Results

- $\bullet~$  Increasing  $\alpha~$  reduces aging ~Y~ and increases FC
- Only small differences between S and  $S_1$  (< 2.5%)



#### Conclusions

Neglecting battery temperature changes in the EMS design leads to an acceptable sub-optimal solution except for extremely aging-biased cost functions

# Battery temperature: Toy Problem

- Motivation: Justify and understand the previous result of model simplification.
- Toy problem considered
  - Cost function:

$$J(u) = \int_0^T \left[\frac{1}{2}(1-\alpha)au^2 + \alpha b\theta_0\theta\right]dt$$

• Dynamics:

$$\dot{\xi} = D - u$$

$$\dot{\theta} = c(D_m - u) - k\theta$$

• Boundary conditions:

$$\xi(0) = \xi_0 \quad \xi(T) = \Delta$$
$$\theta(0) = \theta_0, \theta(T) = \text{free}$$

## Battery temperature: Toy Problem

$$(S_1) \begin{cases} H_1 = \frac{1}{2}(1-\alpha)au^2 + \alpha b\theta_0^2 + \lambda(D-u) \\ \lambda = \frac{(1-\alpha)aE}{T}, \quad E = \int_0^T Ddt - \delta, \quad u_0^* = \frac{E}{T} \\ H = \frac{1}{2}(1-\alpha)au^2 + \alpha b\theta_0\theta + \lambda(D-u) + \mu(c(D_m-u) - k\theta) \\ \lambda = \frac{(1-\alpha)aE}{T} - cp_1\left(1 - \frac{1-e^{-kT}}{kT}\right), \quad p_1 = \frac{\alpha b\theta_0}{k} \\ \mu(t) = p_1\left(1 - e^{-kT}e^{kt}\right) \\ u^* = u_0^* + \frac{cp_1}{(1-\alpha)a}\left(\frac{1-e^{-kT}}{kT} - e^{-kT}e^{kt}\right) \\ J(u_0^*) - J(u^*) = \frac{c^2\alpha^2b^2\theta_0^2}{(1-\alpha)ak^3}\left[\frac{(1-e^{-kT})^2}{2kT} - \frac{1}{4}(1-e^{-2kT})\right] \text{ (small)} \end{cases}$$

The toy model analysis shows that the difference is due to two partially compensating effects

# Catalyst Temperature: Motivation

- Minimizing fuel consumption only without ad-hoc catalyst light-off strategies might not ensure satisfaction of EU regulations
- Catalyst temperature: a key factor to reduce pollutant emissions
- Multi-objective optimal control formulation with minimization of a mixed fuel-emission cost
- Thermal dynamics (engine and catalyst temperatures) considered in the optimal control

### Objective

- Quantify the benefit of including engine and catalyst temperature in the EMS minimizing a trade off between fuel consumption and emissions
- Find a sub-optimal, real-time capable approach

Context Engine temperature

Battery temperature

# Catalyst Temperature: OCP Formulation

• The combined cost function to be minimized is

$$J(u) = \int_0^T \left[ (1 - \alpha) e(\theta_e) c(u, w) + \alpha m_{CO}(u, w, \theta_e, \theta_c) \right] dt$$

- $\alpha$  is a weighting factor to adjust the two criteria
- $m_{CO}(.)$  is CO emission out of the catalyst

$$m_{CO}(u, w, \theta_e, \theta_c) = m_{CO,h}(u, w) f_{CO}(\theta_e) (1 - \eta_{CO}(\theta_c))$$

- e(.) and  $f_{CO}(.)$  are the correction factors of fuel consumption and CO emission with respect to  $\theta_e$
- $\eta_{CO}$  is the catalyst efficiency for CO
- Three state variables, SOC  $\xi$ , lumped engine and catalyst temperatures  $\theta_e, \ \theta_c$ :

$$\dot{\xi}=f(u,w),\quad \dot{\theta}_e=g(u,w,\theta_e),\quad \dot{\theta}_c=k(u,w,\theta_e,\theta_c)$$

Global constraints:

$$\xi(0) = \xi_0, \quad \xi(T) = \xi_f, \quad \theta_e(0) = \theta_c(0) = \theta_0, \quad \theta_e(T) = \text{free}, \quad \theta_c(T) = \text{free}$$

# Catalyst Temperature: OCP Solution

• Offline solution using PMP

 $H(u,w,\theta_e,\theta_c,\lambda,\mu,\rho) = L(u,w,\theta_e,\theta_c) + \frac{\lambda}{f}(.) + \frac{\mu}{g}(.) + \frac{\rho}{k}(.)$ 

- $\bullet\,$  Initial values  $\lambda(0),\,\mu(0)$  and  $\rho(0)$  found by shooting methods
- Looking for a methodology to include catalyst temperature in real-time EMS
- Proposed solution: Control model (Hamiltonian) simplification

# Catalyst Temperature: OCP Results and Comparison

• Calculation of the control using:

$$S : u^{*}(t) = \arg \min_{u \in U^{ad}} \left[ L(u, w, \theta_{e}, \theta_{c}) + \lambda f(.) + \mu g(.) + \rho k(.) \right]$$
  

$$S_{1} : u^{*}_{1}(t) = \arg \min_{u \in U^{ad}} \underbrace{\left[ L(u, w, \theta_{e} = \theta_{e,h}, \theta_{c}) + \lambda f(.) + \rho k(.) \right]}_{H(u, w, \theta_{e} = \theta_{e,h}, \theta_{c}, \lambda, \mu(t) = 0, \rho)}$$
  

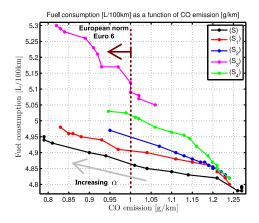
$$S_{2} : u^{*}_{2}(t) = \arg \min_{u \in U^{ad}} \underbrace{L(u, w, \theta_{e} = \theta_{e,h}, \theta_{c} = \theta_{c}(0)) + \lambda f(.)}_{H(u, w, \theta_{e} = \theta_{e,h}, \theta_{c} = \theta_{c}(0), \lambda, \mu(t) = 0, \rho(t) = 0)}$$

• Two additional heuristic strategies, neglecting the adjoint states:

$$S_{3} : u_{3}^{*}(t) = \arg \min_{u \in U^{ad}} \underbrace{\left[ L(u, w, \theta_{e}, \theta_{c}) + \lambda f(.) \right]}_{H(u, w, \theta_{e}, \theta_{c}, \lambda, \mu(t) = 0, \rho(t) = 0)}$$

$$S_{4} : u_{4}^{*}(t) = \arg \min_{u \in U^{ad}} \underbrace{\left[ L(u, w, \theta_{e} = \theta_{e,h}, \theta_{c}) + \lambda f(.) \right]}_{H(u, w, \theta_{e} = \theta_{e,h}, \theta_{c}, \lambda, \mu(t) = 0, \rho(t) = 0)}$$

## Catalyst Temperature: OCP Results and Comparison



### Conclusions

• Neglecting temperatures changes in the EMS design leads to a acceptable sub-optimal solutions except for extremely emission-biased cost functions

## Theoretical Justification: Approach

### Next step

- Justify from theoretical viewpoint (general case) the model simplifications presented (engine, battery and catalyst temperatures).
- We want to solve the following optimal control problem:

$$(P_{\varepsilon}) \begin{cases} \min_{u \in U^{ad}} J_{\varepsilon}(u) = \int_{0}^{T} l_{\varepsilon}(x_{\varepsilon}, u) dt, \\ \dot{x}_{\varepsilon} = f_{\varepsilon}(x_{\varepsilon}, u), \\ x_{\varepsilon}(0) = x_{0}, \end{cases} \qquad \boxed{x = [\xi, \ \theta_{e}]}$$

$$l_{\varepsilon}(x_{\varepsilon}, u) = c(u, w)e(\theta_e)$$

• By solving the following simplified (nominal) problem:

$$(P_0) \begin{cases} \min_{u \in U^{ad}} J_0(u) = \int_0^T l_0(x_0, u) dt, \\ \dot{x}_0 = f_0(x_0, u), \\ x_0(0) = x_0, \end{cases}$$

$$l_0(x, u) = c(u, w)$$

## Theoretical Justification: Result

#### Main result

Under some assumptions, we prove that  $\exists K > 0$  st:

$$J_{\varepsilon}(u_0) - J_{\varepsilon}(u_{\varepsilon}) \le K\varepsilon^2$$

• How can we determine if K is small? Counter example

### Table: Estimation of K based on nominal solution

	Thermal management	Eco-driving
$K_{est}/K_{num}$	11	26
$\Delta J/J_{\varepsilon}(u_0)$	1 %	90 %
$K_{est}/J_{\varepsilon}(u_0)$	0.105	19.62
Sub-optimality	Acceptable	Not Acceptable

# Conclusions and perspectives

### • Conclusions:

- Hamiltonian simplification leads to acceptable sub-optimality for thermal-management problems
- A priori calculation of K can be used to predict the sub-optimality induced by control model (Hamiltonian) simplification

### • Current work:

- Find a solution to adapt the adjoint states as a function of the temperature measurements in real-time
- Experimental validation of the results concerning catalyst temperature (Diesel hybrid)

### • Papers:

- D. Maamria, F. Chaplais, N. Petit and A. Sciarretta. Numerical Optimal Control As a Method to Evaluate the Benefit of Thermal Management in Hybrid Electric Vehicles . in Proc. of the 2014 IFAC World Congress.
- D. Maamria, F. Chaplais, N. Petit and A. Sciarretta. On the impact of model simplification in input constrained optimal control: application to HEV energy-thermal management. IEEE-CDC 2014 (To appear).
- D. Maamria, F. Chaplais, N. Petit and A. Sciarretta. Comparison of several strategies for HEV energy management system including engine and catalyst temperatures. ACC 2015 (Submitted).